

*Massively parallel methods for data-driven modelling and simulation in computational mechanics*  
*Virtual workshop*  
*September 23, 2020.*

# A fairly priced, unfitted spline image-based model to assist Digital Image Correlation

Ali Rouwane<sup>1,2</sup>, Robin Bouclier<sup>1,2</sup>, Jean-Charles Passieux<sup>1</sup> and Jean-Noël Périé<sup>1</sup>

rouwane@insa-toulouse.fr ; alirouwane@yahoo.com

1. Institut Clément Ader (ICA), Université de Toulouse, CNRS/INSA/Mines Albi/UPS/ISAE, Toulouse, France
2. Institut de Mathématiques de Toulouse (IMT), Université de Toulouse, CNRS/UT1/UT2/UT3/INSA, Toulouse, France

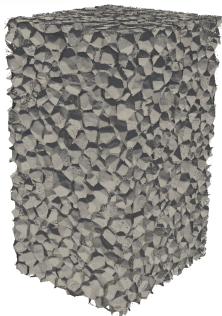


# Introduction

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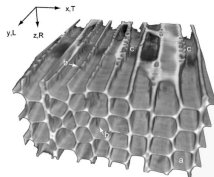
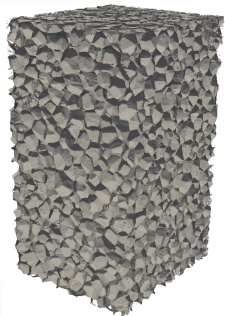
Estimation of displacement and strain fields at the cellular scale of materials with complex cellular microstructures using Digital Image Correlation.

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(a) Polymetacrylid foam (Rohacell-51) image obtained from X-ray micro-tomography  $\times$

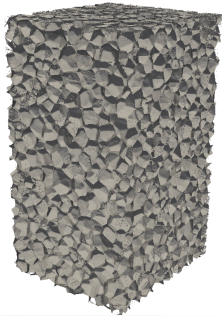
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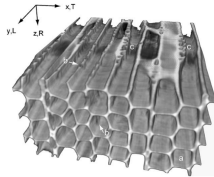
(a) Polymetacrylid foam (Rohacell-51) image obtained from X-ray micro-tomography ✕

(b) Wood cell image obtained from X-ray micro-tomography Forsberg et al. [2008] ✕

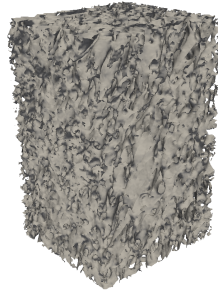
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(c) Cattle bone image obtained from MRI Benoit et al. [2009] ✕

Estimation of displacement and strain fields at the cellular scale of materials with complex cellular microstructures using Digital Image Correlation.

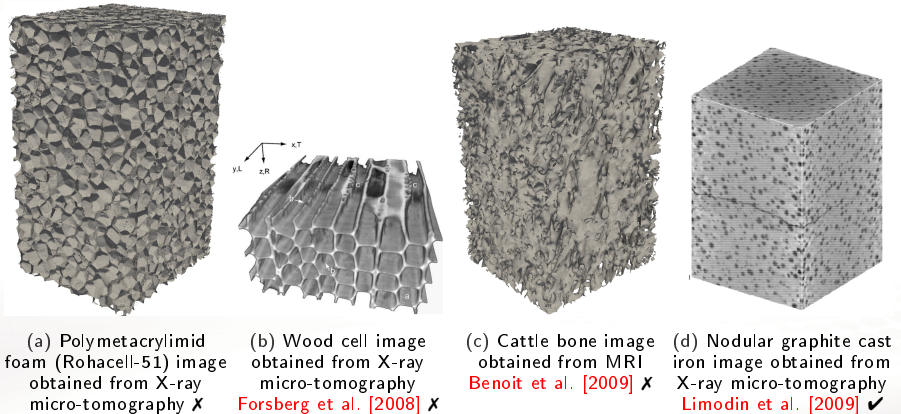


Figure 1: Example of textures of complex materials.

More void than material → Poor texture makes the optimization problem difficult without using regularization schemes.

- Develop a 2D DIC algorithm that allows to estimate the displacement behind transformation

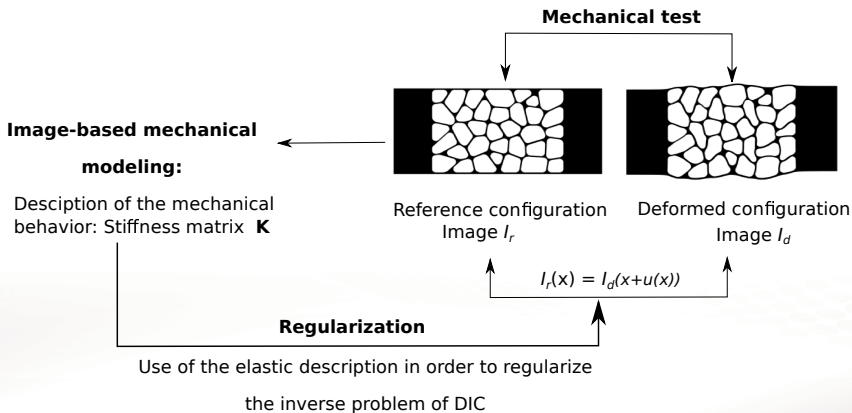


Figure 2: Summary of the methodology. Elastic regularization of DIC.

1. Building the stiffness operator  $\mathbf{K}$  on the ROI using a fairly-priced image-based mechanical model



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  - o Presentation a geometry analysis study for fine-tuning the model's integration parameters
  - o Presentation of a mechanical convergence study and confrontation with low order FEM image-based models.
2. Use of the built stiffness operator for the regularization of Digital Image Correlation

- Deforming images by moving the control points of a B-spline control grid

(a)  $C^0$  linear B-splines (same as linear lagrange functions)

(b)  $C^1$  quadratic B-splines.

(c)  $C^2$  cubic B-splines.

Figure 3: Image deformation with B-splines of  $C^{p-1}$  regularity at the element boundaries.

- Acquired image

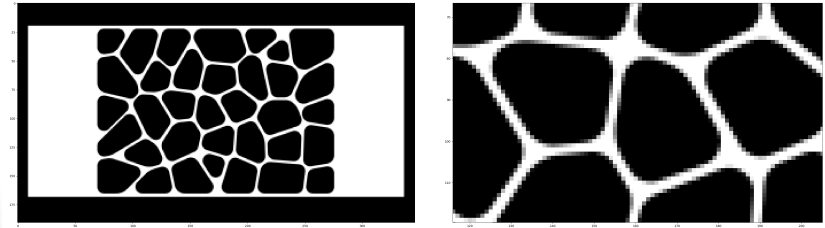
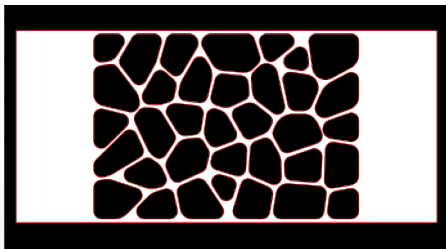


Figure 4: Image acquisition of a 2d sample with a complex geometry

- Level-set description of the geometry of two-phase materials:
  - Evolution level-sets based on the convection-diffusion equation [Chan and Vese \[2001\]](#); [Bernard et al. \[2008\]](#)
  - Iso-value of a smooth physical representation of the target image [Verhoosel et al. \[2015\]](#)



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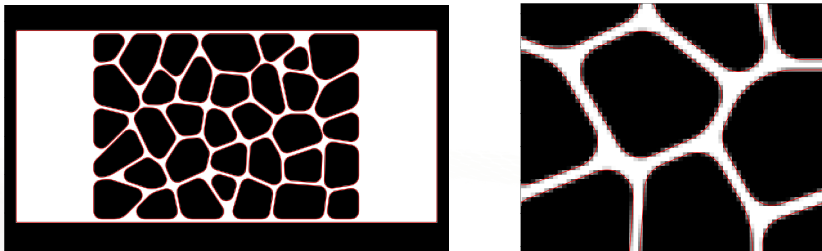


Figure 5: Level-set description of the physical domain.



- Embedding the image domain in a rectangular mesh Parvizian et al. [2007]; Schillinger and Ruess [2015]; Verhoosel et al. [2015]

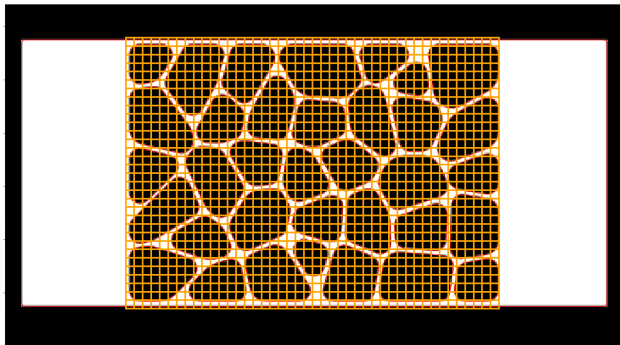


Figure 6: Embedding of the level-set geometry using a B-spline mesh (here the parametric space is equal to the physical space).

- Integrating only on the physical domain. The level-set geometry is approximated by a quad-tree integration scheme Düster et al. [2008]; Schillinger and Ruess [2015] with a closure tessellation scheme Verhoosel et al. [2015].

# Construction of a fairly-priced image-based mechanical model with the Finite Cell Method

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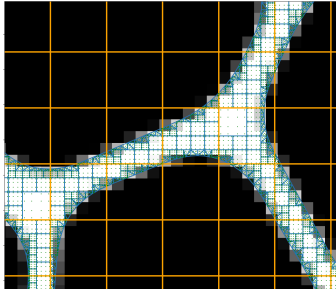


Figure 7: Image acquisition of an image of complex geometry

- Penalization of the stress tensor in the fictitious domain.

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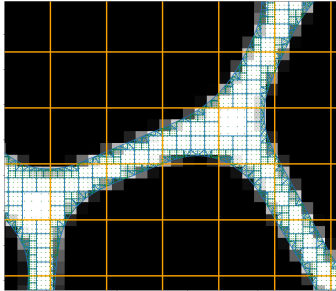


Figure 7: Image acquisition of an image of complex geometry

- Penalization of the stress tensor in the fictitious domain.
- Other closure integration techniques
  - Moment fitting methods Abedian et al. [2013]; Müller et al. [2013]; Joulaian et al. [2016]
  - Smart boundary conforming octrees Kudela et al. [2016]

# Construction of a fairly-priced image-based mechanical model with the Finite Cell Method

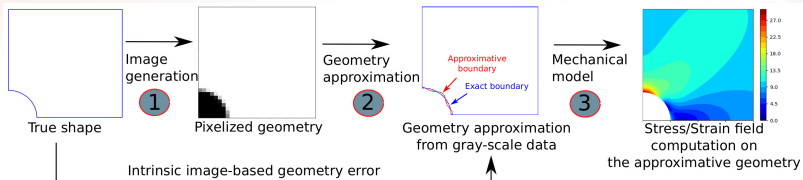


Figure 8: Summary of the different steps of the construction of a mechanical digital image-based model.

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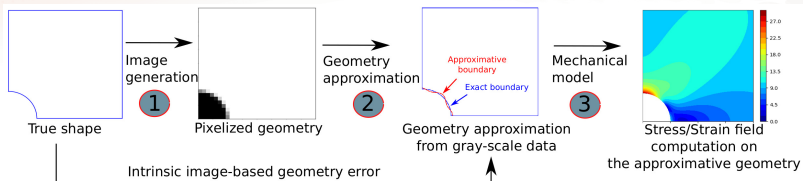


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- Geometry error analysis

# Construction of a fairly-priced image-based mechanical model with the Finite Cell Method

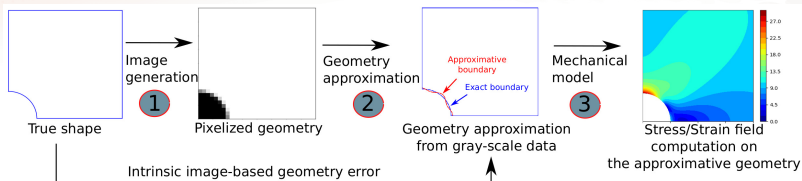


Figure 8: Summary of the different steps of the construction of a mechanical digital image-based model.

- Geometry error analysis

- Intrinsic geometry error:

$$E = \frac{|\tilde{A} - A|}{A} \quad (1)$$

- Total geometry error:

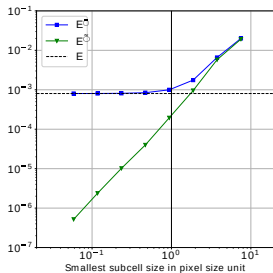
$$\bar{E} = \frac{|A_a - A|}{A} \quad (2)$$

- Domain integration error:

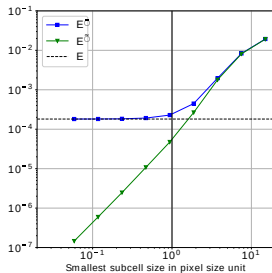
$$\check{E} = \frac{|A_a - \tilde{A}|}{\tilde{A}} \quad (3)$$

where  $A_a$  is the approximation of the area bounded by the level-set using the quad-tree scheme.  $\tilde{A}$  and  $A$  are respectively the area of the level-set geometry and the exact area of the reference geometry

- Geometric error evolution:



(a)  $30 \times 30$  pixels in the image.

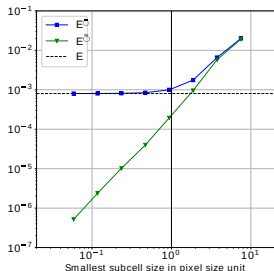


(b)  $60 \times 60$  pixels in the image.

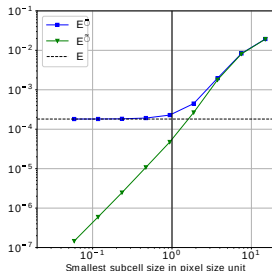
Figure 9: Evolution of the errors  $\bar{E}$  and  $\tilde{E}$  with respect to the size of the smallest sub-cell in pixel size units for the two-dimensional test case.



- Geometric error evolution:



(a)  $30 \times 30$  pixels in the image.



(b)  $60 \times 60$  pixels in the image.

Figure 9: Evolution of the errors  $\bar{E}$  and  $\tilde{E}$  with respect to the size of the smallest sub-cell in pixel size units for the two-dimensional test case.

- A sufficient quad-tree level can be set so that the smallest sub-cell size is approximately equal to the pixel size.

$$l = \left\lceil \frac{1}{2} \log_2 \left( \frac{n_x n_y}{n_x^e n_y^e} \right) \right\rceil. \quad (4)$$

# Comparison of the FCM image-based model to other Finite Element-based image models

We compare with three other lower finite element methods for computing the mechanical solution.

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**1. Voxel based model:** Convert the connectivity of the binary image into a  $Q_4$  finite element mesh [Ulrich et al. \[1998\]](#).

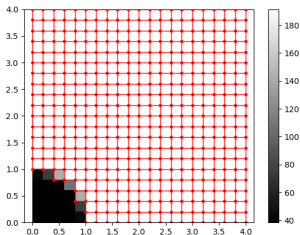


Figure 10: Finite element mesh of a binary image.

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**2. Marching squares algorithm:** Extraction of a linear boundary and triangular meshing  
 Lorensen and Cline [1987]; Frey et al. [1994]; Ulrich et al. [1998].

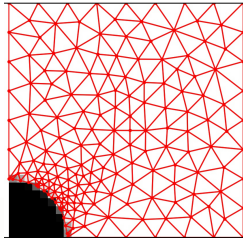


Figure 11: Extraction of a Finite element mesh using the marching squares algorithm.

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**3. Gray-level dependant mechanical properties** : Assign each pixel a mechanical property that depends on its gray-level value. Example of a linear mechanical law for two materials.

$$E(v) = \frac{v - v_{min}}{v_{max} - v_{min}} E_{max} + \frac{v_{max} - v}{v_{max} - v_{min}} E_{min}$$

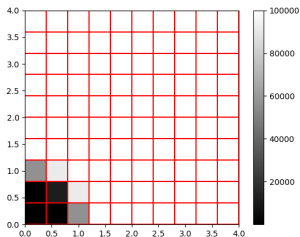


Figure 12: Mechanical properties ranging from 1Pa to  $10^5$  Pa.

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**4. Level-set based FCM with a triangular tessellation closure:** Fictitious domain method on a linear triangular geometry approximating a continuous geometry defined by a level-set.

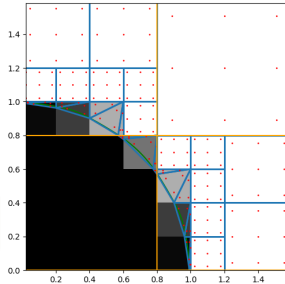
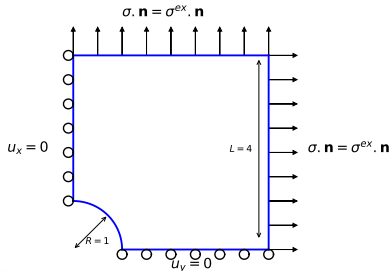
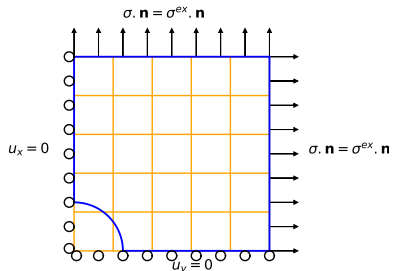


Figure 13: Fictitious domain method on a binary geometry.

# Comparison of the FCM image-based model to other Finite Element-based image models

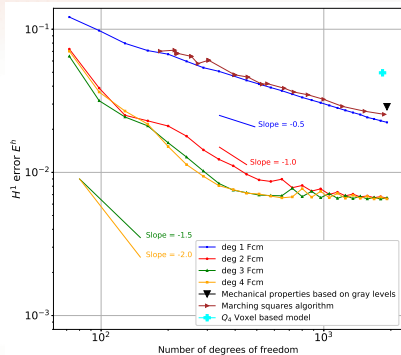


(a) Mechanical problem definition.

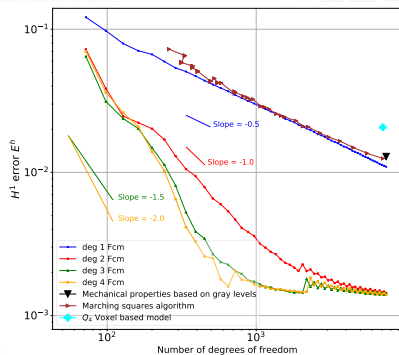


(b) An example of the embedding B-spline elements with the corresponding boundary conditions.

Figure 14: Mechanical problem definition: elastic plate with a quarter hole. The definition of  $\sigma^{ex}$  can be found in [Sadd \[2009\]](#)



(a)  $30 \times 30$  pixels in the image.



(b)  $60 \times 60$  pixels in the image.

Figure 15: Evolution of the error in energy norm under mesh refinement.



The suggested methodology aims at estimating displacement fields at the cellular scale by solving the DIC problem:

$$I_r(x) = I_d(x + u(x)) \quad (5)$$

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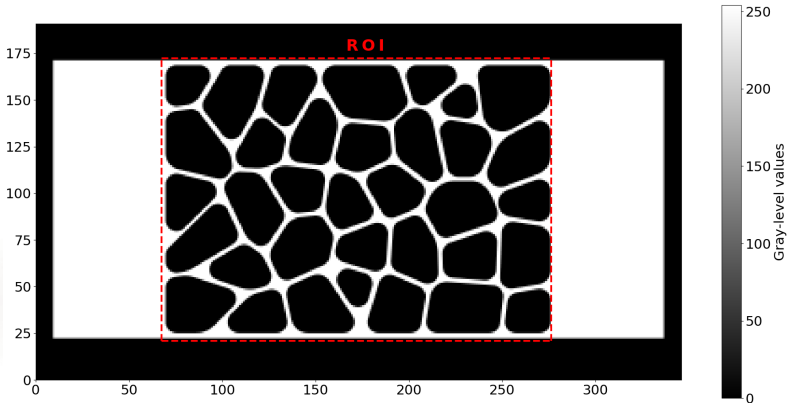


Figure 16: Image  $I_r$

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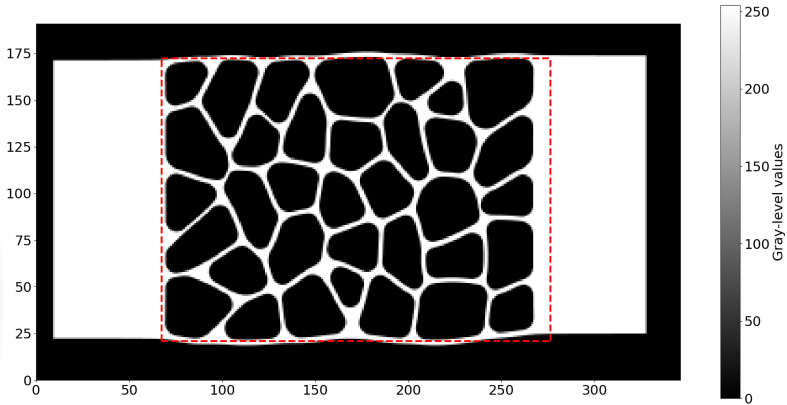


Figure 17: Image  $I_d$

- Displacement searched in a subspace of  $L^2(\Omega)$  spanned by as set of basis functions:

$$u(x, y) = \mathbf{N}(x, y)\mathbf{u} \quad (6)$$

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- Problem (5) is changed into the minimization of the squared  $L^2$  norm

$$S(\mathbf{u}) = \frac{1}{2} \int_{\Omega} (I_r(x, y) - I_d((x, y) + \mathbf{N}(x, y)\mathbf{u}))^2 dx dy \quad (7)$$

- Multi-level displacement estimation using a  $Q_4$  finite element mesh.

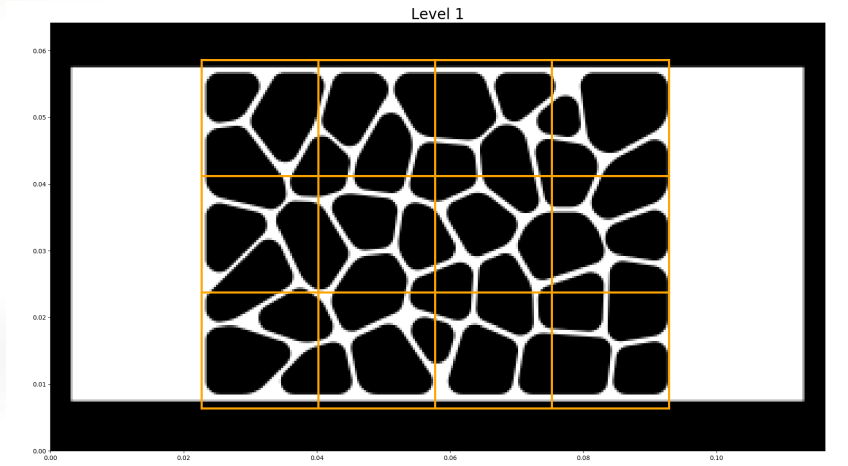


Figure 18: Level 1 of refinement

- Multi-level displacement estimation using a  $Q_4$  finite element mesh.

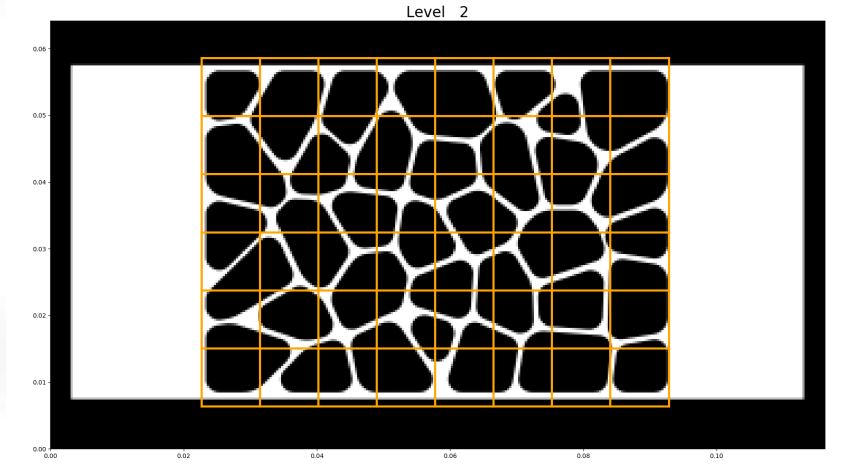


Figure 19: Level 2 of refinement

- Multi-level displacement estimation using a  $Q_4$  finite element mesh.

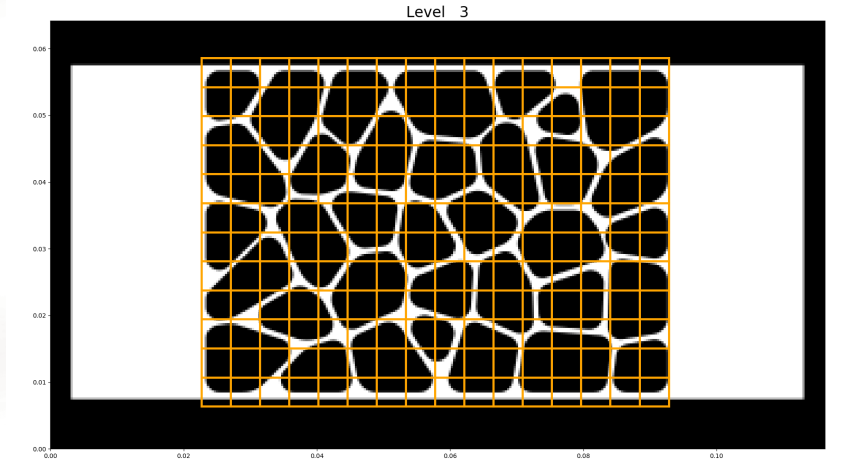
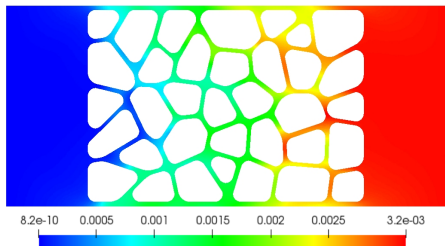
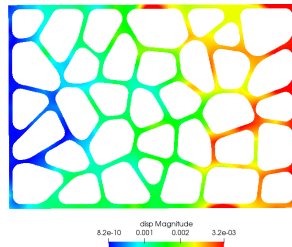


Figure 20: Level 3 of refinement



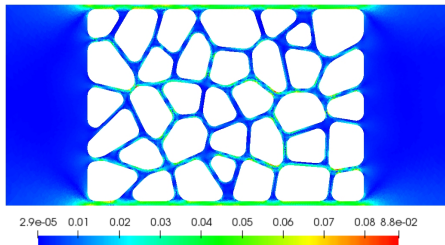


(a) Finite element reference displacement field.

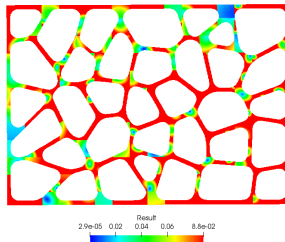


(b) Estimated displacement field.

Figure 21: Displacement field comparison



(a) Finite element reference displacement field.



(b) Estimated displacement field.

Figure 22: Strain field comparison

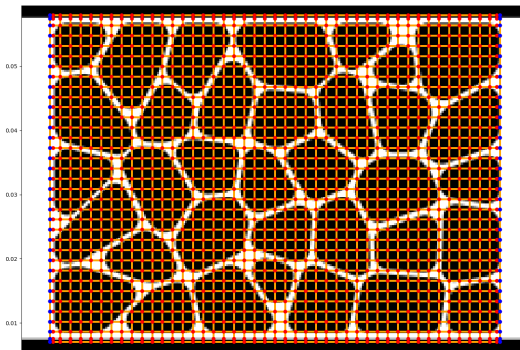


Figure 23: B-spline mesh displayed the grid of B-spline control points.

- Equilibrium gap regularization Réthoré et al. [2009]

$$M(\mathbf{u}) = \frac{1}{2} \|\mathbf{K}\mathbf{u} - \mathbf{f}\|_2^2$$

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- First order gradient Tikhonov regularization

$$T(\mathbf{u}) = \frac{1}{2} \|\mathbf{D}_T \mathbf{L}\mathbf{u}\|_2^2 \quad (9)$$

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$$T(\mathbf{u}) = \frac{1}{2} \|\mathbf{D}_T \mathbf{L}\mathbf{u}\|_2^2 \quad (9)$$

- The optimization functional

$$\arg \min_{\mathbf{u} \in \mathbb{R}^{2n}} [S(\mathbf{u}) + \lambda_M M(\mathbf{u}) + \lambda_T T(\mathbf{u})], \quad (10)$$

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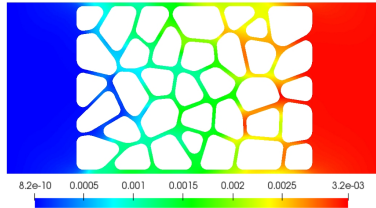
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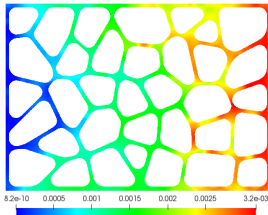
- Problem (10) is solved with a modified Gauss-Newton scheme (see e.g. Passieux and Bouclier [2019] for more details on the optimization scheme).
  - Leads to an iterative scheme where a SPD linear system is solved at each iteration  $\longrightarrow$  ill-conditioned Hessian due to the poor conditioning of the stiffness matrix



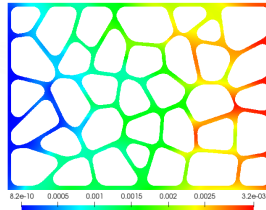
# Numerical results: comparison of the euclidian norm of the displacement



Reference finite element simulation

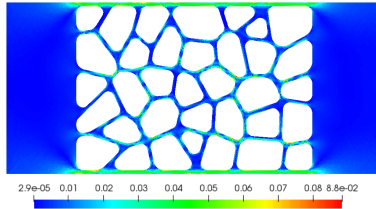


Tikhonov regularization

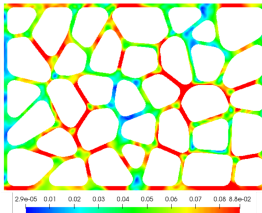


Equilibrium gap regularization

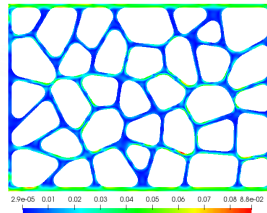
# Numerical results: comparison of the Von-Mises strain norm of the displacement



Reference finite element simulation

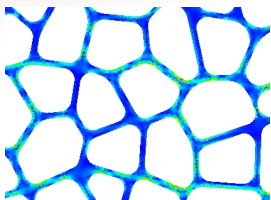


Tikhonov regularization

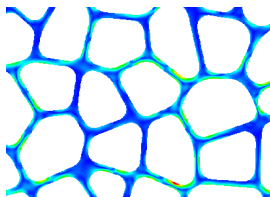


Equilibrium gap regularization

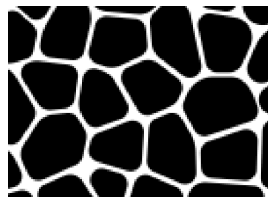
## Numerical results: comparison of the Von-Mises strain norm of the displacement



(a) Strain norm of the finite element simulation.



(b) Strain norm of the registered solution.



(c) Reference image  $I_r$ .

Figure 24: Zoom on a region in the ROI.

	$P(u_x)$ (pixels)	$P(u_y)$ (pixels)	$P(\varepsilon_{xx})$	$P(\varepsilon_{yy})$	$P(\varepsilon_{xy})$
Standard multi-level DIC	$4.5 \times 10^{-1}$	$1.9 \times 10^{-1}$	$7.2 \times 10^{-1}$	1	$4 \times 10^{-1}$
Tikhonov regularization	$1.6 \times 10^{-1}$	$1.1 \times 10^{-1}$	1.4	1.4	$4.1 \times 10^{-1}$
Mechanical regularization	$2 \times 10^{-2}$	$3 \times 10^{-2}$	$3.8 \times 10^{-2}$	$1 \times 10^{-2}$	$3.5 \times 10^{-3}$

Table 1: Precision of the measurements in terms of displacement and strain fields.

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- Perspectives
  - Generalization of the method to 3D and application to Digital Volume Correlation.
    - ⊖ Until know direct sparse linear solvers are used (SuperLU)
    - ⊖ Currently working on solving using other libraries (such as MUMPS)

## ● Conclusion

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- We analysed these geometric and mechanical errors and proposed a pragmatic rule to set the Finite Cell Method parameters.
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## ● Perspectives

- Generalization of the method to 3D and application to Digital Volume Correlation.
  - ⊖ **Until know direct sparse linear solvers are used (SuperLU)**
  - ⊖ **Currently working on solving using other libraries (such as MUMPS)**
- Application with real *in-situ* mechanical tests using computed Micro-tomography.



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The constitutive behavior law is modified by considering the penalized stress tensor defined by:

$$\sigma_\alpha(x, y) = \alpha(x, y)\sigma, \quad (11)$$

with

$$\alpha(x, y) = \begin{cases} \alpha_p = 1 & \forall (x, y) \in \Omega_p \\ \alpha_f = 10^{-q} \ll 1 & \forall (x, y) \in \Omega_f \end{cases} . \quad (12)$$

Instead of performing  $\mathbf{K}_{\Omega_f}(\alpha_f) + \mathbf{K}_{\Omega_p}(\alpha_p)$  we assemble two stiffness matrices (one homogeneous on all elements) and one only on the integration domain

$$\mathbf{K} = \mathbf{K}_\Omega(\alpha_f) + \mathbf{K}_{\Omega_p}(\alpha_p - \alpha_f). \quad (13)$$

The resolution of the regularized non-linear least squares problem (10) is performed using the following descent scheme:

$$\mathbf{u}^{(k+1)} = \mathbf{u}^{(k)} + \mathbf{d}^{(k)}, \quad (14)$$

where  $\mathbf{d}^{(k)}$  is the solution of the following Gauss-Newton system:

$$\left( \mathbf{H}_S(\mathbf{u}^{(k)}) + \lambda_M \mathbf{H}_M(\mathbf{u}^{(k)}) + \lambda_T \mathbf{H}_T(\mathbf{u}^{(k)}) \right) \mathbf{d}^{(k)} = - \left( \nabla S(\mathbf{u}^{(k)}) + \lambda_M \nabla M(\mathbf{u}^{(k)}) + \lambda_T \nabla T(\mathbf{u}^{(k)}) \right) \quad (15)$$

and where  $\mathbf{H}_S$  is an approximation using only first-order derivatives of the Hessian matrix of  $S$ .  $\mathbf{H}_M$  and  $\mathbf{H}_T$  are respectively the Hessian matrices of  $M$  and  $T$  and  $\nabla_S, \nabla_M, \nabla_T$  are respectively the gradient vectors of  $S, M$  and  $T$ . The definition of these six operators is given by equations (16), (17), (18) and (19), see below:

$$\nabla S(\mathbf{u}^{(k)}) = - \int_{\Omega} \left( I_r(x, y) - I_d \left( (x, y) + \mathbf{N}(x, y) \mathbf{u}^{(k)} \right) \right) \mathbf{N}(x, y)^T \nabla I_r(x, y) dx dy; \quad (16)$$

$$\nabla M(\mathbf{u}^{(k)}) = \mathbf{K}^T \mathbf{D}_M^T \mathbf{D}_M \mathbf{K} \mathbf{u}^{(k)}, \quad \nabla T(\mathbf{u}^{(k)}) = \mathbf{L}^T \mathbf{D}_T^T \mathbf{D}_T \mathbf{L} \mathbf{u}^{(k)}; \quad (17)$$

$$\mathbf{H}_S(\mathbf{u}^{(k)}) = \int_{\Omega} \mathbf{N}(x, y)^T (\nabla I_d) \left( (x, y) + \mathbf{N}(x, y) \mathbf{u}^{(k)} \right)^T (\nabla I_d) \left( (x, y) + \mathbf{N}(x, y) \mathbf{u}^{(k)} \right) \mathbf{N}(x, y) dx dy; \quad (18)$$

$$\mathbf{H}_M = \mathbf{K}^T \mathbf{D}_M^T \mathbf{D}_M \mathbf{K}, \quad \mathbf{H}_T = \mathbf{L}^T \mathbf{D}_T^T \mathbf{D}_T \mathbf{L}. \quad (19)$$

$$\mathcal{T}(u) = \frac{1}{2} \int_{\Omega} \|\nabla u_x\|_2^2 + \|\nabla u_y\|_2^2 dx dy = \int_{\Omega} \left( \frac{\partial u_x}{\partial x} \right)^2 + \left( \frac{\partial u_x}{\partial y} \right)^2 + \left( \frac{\partial u_y}{\partial x} \right)^2 + \left( \frac{\partial u_y}{\partial y} \right)^2 dx dy. \quad (20)$$

The discrete form directly coming from  $\mathcal{T}$  is given by:

$$\tilde{\mathcal{T}}(\mathbf{u}) = \frac{1}{2} \int_{\Omega} \sum_{i=1}^4 \|\mathbf{L}_i(x, y)\mathbf{u}\|^2 dx dy = \frac{1}{2} \mathbf{u}^T \left( \int_{\Omega} \sum_{i=1}^4 \mathbf{L}_i^T(x, y)\mathbf{L}_i(x, y) dx dy \right) \mathbf{u} = \frac{1}{2} \mathbf{u}^T \mathbf{L}\mathbf{u}, \quad (21)$$

where  $\mathbf{L}_i$  are first order partial differential operators.  $\mathbf{L}$  is called the Tikhonov linear operator. In order to properly select the DOF where the Tikhonov regularization will be applied, we will eventually consider a slightly different discrete cost function, based on the euclidean norm of the action of the Tikhonov operator instead of the scalar product:

$$\tilde{\mathcal{T}}(\mathbf{u}) = \frac{1}{2} \|\mathbf{L}\mathbf{u}\|_2^2. \quad (22)$$